ASSESSMENT OF FUEL CELLS AND PRESSURIZED WATER NUCLEAR RE-ACTORS FOR SUBMARINE PROPULSION

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by

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B.S. in O.E. U.S. Naval Academy (1972)

Submitted in partial fulfillment of the requirements for the Degree of Ocean Engineer and the Degree of Master of Science in Mechanical Engineering

at the

MASSACHUSETTS INSTITUTE OF TECHNOLOGY

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Submitted to the Department of Ocean Engineering and Mechanical Engineering on 17 December 1976, in partial fulfillment of the requirements for the degree of Ocean Engineer and the degree of Master of Science in Mechanical Engineering at the Massachusetts Institute of Technology.

ABSTRACT

Nuclear reactors have been powering U.S. Navy submarines for over twenty years with basically the same power plant, the pressurized water reactor. The hydrogen-oxygen fuel cell, with technology learned from the Space Program, offers a possible alternate for selected missions. Employing a math model based on existing nuclear powered submarines, but modified to encompass the fuel cell submarine, the effects of the two power plant types on payload carrying capabilities, maximum velocities, endurances, and capital cost are studied.

The fuel cell submarine is feasible at low vehicle sizes (1000 tonnes) at low shaft horsepower (5000 HP). The nuclear submarine is not. When comparing payload carrying capabilities, the two power plant types offer similar capabilities for endurance values of 2000 and 4000 miles for the fuel cell submarines. However, in the capital cost area, the fuel cell submarine cost is 65% to 85% of the nuclear submarine cost with savings of 18 to 60 million dollars. While the fuel cell submarine cannot compete with nuclear submarines in endurance, there are some missions where fuel cell submarines can do the job, at lower costs.

Thesis Supervisor: Philip Mandel

Title: Professor of Naval Architecture



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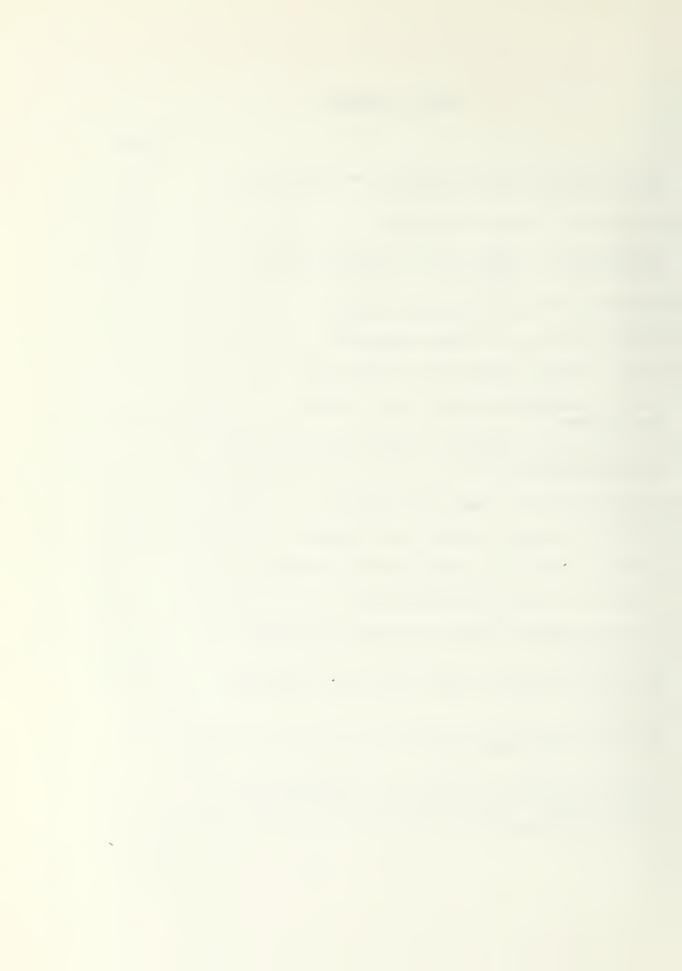
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DEFINITION OF SYMBOLS USED

| Symbol | <u>Definition</u> | <u>Units</u> |
|-----------------|---|---|
| С | Capital Cost | $$ \times 10^{-3}$ |
| ΔC _f | Additional Roughness Coefficient | |
| C _p | Prismatic Coefficient | |
| D | Submarine Diameter | feet |
| D[I] | Vehicle Mass | TONNES |
| E[I] | Endurance | Nautical miles |
| G | Vertical Center of Gravity Fraction of Hull Diameter | |
| Н | Total Labor Hours | Man hours |
| н1←→н9 | Group Labor Man Hours | Man hours |
| k | Proportional Constant | HP/[(LTONS) ^{2/3} (KTS) ³] |
| K | Proportional Constant | $HP/[(LTONS)^{2/3}$ $(KTS)^3$ |
| K1 | Fuel Cell Power | KM |
| L(I) | Non-Propulsive Electric Load | KW |
| М | Total Material Cost | \$ |
| M1-M9 | Group Material Cost | \$ |
| N1 | # Enlisted | Men |
| N2 | # CPO's | Men |
| N3 | # Officers | Men |
| N4 | Total Personnel | Men |
| Pl | Payload Mass | TONNES |
| P2 | Payload Volume | m ³ |



| P9 | Correction Factor | |
|---|-------------------------------|--------------------|
| Rl | Payload Mass/Vehicle Mass | |
| R2 | Payload Volume/Vehicle Volume | |
| R3 | Payload Density | kg/m ³ |
| RPM | Shaft Revolutions/minutes | rev/min |
| SHP, S1 | Shaft Horsepower | HP |
| S[I] | Shaft Horsepower/1000 | HP |
| т | Total Vertical Moment | FT TONNES/FT |
| т1-т9 | Group Vertical Moment | FT TONNES/FT |
| t _D | Test Depth | feet |
| V | Total Volume | m ³ |
| v1-v9, v _p , v _{pr} | Group, Personnel, Provision | m ³ |
| W | Volume Total Mass | TONNES |
| w1-w9, w _p , w _{pr} | Group, Personnel, Provision | TONNES |
| х | Mass Vehicle Volume | m ³ |
| Y | Velocity | KNOTS |
| z1-z7, z _p , z _p r | Vertical Center of Gravity | feet |
| ^ζ 2 ^{-ζ} 7' ^ζ pr | Group Provision Density | kg/m ³ |
| ζ _s | Density of Steel | lb/ft ³ |
| σу | Yield Strength | psi |



CHAPTER I

INTRODUCTION

Prior to the launching of the nuclear powered submarine, USS NAUTILUS, in 1954, all combat submarines were powered by a combination of diesel generators and batteries. When unable to snorkel, the submarine could only remain submerged for a limited amount of time. With the advent of nuclear power, submarines were freed from the surface, and became true submarines, limited in submergence time by personnel requirements rather than by power requirements. With the exception of the diesel-electric powered BARBEL class, all combat submarines built for the U.S. Navy since 1954 have been powered by nuclear reactors. There are over 100 nuclear powered submarines in the U.S. Navy today, with power plants essentially the same as the NAUTILUS.

One of the benefits of the space program was the advancement in fuel cell technology. Fuel cells were used in both the GEMINI and APOLLO space missions and fuel cells will provide electric power in the Space Shuttle. Fuel cells consume oxygen and hydrogen and produce direct current electricity and water. Based on the performance of fuel cells in the space program, and the advantages in weight and volume, fuel cells offer an interesting alternative to nuclear power for submarines of modest size and endurance. One of the DSRV's, a U.S. Navy submarine rescue submersible, is being equipped with a fuel cell to extend its endurance over that possible with batteries of the current technology.



I.A. Methodology

Independent inputs in this study are vehicle mass, configuration, and type and size of power plant. Outputs include maximum speed, payload mass and volume, limiting center of gravity location, endurance range at cruising speed, and capital costs. (See Figure 1.)

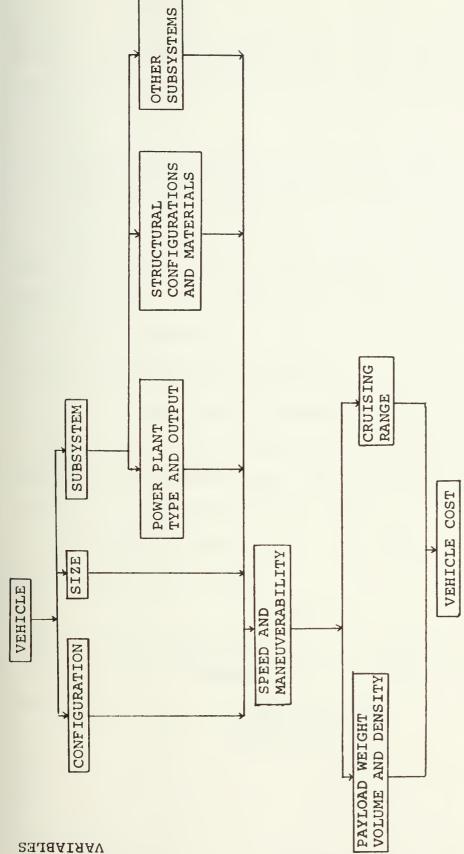
I.A.l. Vehicle Mass

Vehicle mass was limited to four inputs: 1000, 2000, 5000, and 10000 tonnes. This range includes all existing submarines in the U.S. Navy.

I.A.2. Configuration

The vehicle was constrained to a body of revolution, single screw submarine with a prismatic coefficient of 0.70 and a length to diameter ratio of 9.0. The maximum horse-power per shaft was limited to 100,000 H.P. Therefore based on these two constraints, the input power was limited to 100,000 H.P. As the results will show, however, submarines with greater than 100,000 H.P. were infeasible for other reasons. The maximum operating depth was fixed at 400 m. (1312 ft.) and the crush depth at 600 meters (1969 ft.). The operating depth was fixed in order to calculate a respresentative hull mass. Payload masses would increase with decreasing depth. HY-80 steel was used for the hull material. The propeller was constrained to being a fully submerged, fixed pitch propeller with a diameter two-thirds the hull





INDEPENDENT

METHODOLOGY FIGURE 1



diameter. The vehicle density was fixed by bouyancy requirements at the density of sea water, 1025 kg/m^3 (64 1b/ft^3). This configuration is typical of nuclear powered attack submarines.

I.A.3 Types and Power Outputs of Power Plants

There were two types of power plants studied; the pressurized water nuclear reactor and the hydrogen-oxygen fuel cell. The power output inputted into this study range from 5000 to 400,000 H.P. However, all power plants above 100,000 H.P. proved infeasible.

I.A.3.a. Nuclear Power Reactors

Nuclear reactors generate heat by the fission of uranium. There are four basic types of nuclear reactors generating electricity today: the pressurized water reactor (PWR), the boiling water reactor (BWR), gas cooled reactor, and liquid metal cooled reactor. See Table 1 for a comparison of reactor core types.

TABLE 1
Comparison of Representative Power Densities for a 500 MW
(Thermal) Reactors (Ref. 1)

| Reactor Type | Representative Power Density (MW(Thermal)/ft ³) | Area Volume (ft ³) |
|--------------|---|-----------------------------------|
| PWR | 2.5 | 200 |
| BWR | 0.8 | 650 |
| Gas Cooled | 0.25 | 2000 |
| Liquid Metal | 25.0 | 20 |



While the liquid metal reactor core is the smallest due to high coolant heat transfer coefficients, other problems concerning radioactivity requiring heavy shielding, chemical activity, corrosion and thermal shock make this power plant a difficult choice for marine propulsion. The USS SEAWOLF, a US. Navy submarine built in 1957, was powered by a liquid metal cooled reactor, but it was replaced after two years by a PWR.

The BWR is less suitable for marine use due to design constraints on the control rods, necessitating that the reactor vessel be mounted relatively high in the ship. Another potential difficulty in marine use of a BWR plant is reactivity changes associated with void movement and volume change due to ship motions caused by shock, impact, and seaway forces. (Ref. 1)

The PWR is the best choice for marine nuclear propulsion today. It consists of a primary and a secondary system. The primary system deals with equipment which contacts the main coolant of the reactor. It consists of the reactor core, steam generators, a pressurizer to absorb transients, coolant pumps to remove heat from the core and transport it to the steam generator, and the necessary piping and shielding. The steam generator transfers the heat from the coolant to the steam of the secondary system.

The secondary system is a basic Rankine cycle steam power plant, with the boiler replaced by a steam generator. The output of the steam generator is saturated steam at 700-1000 psia and 500-550°F. (See Figure 2)



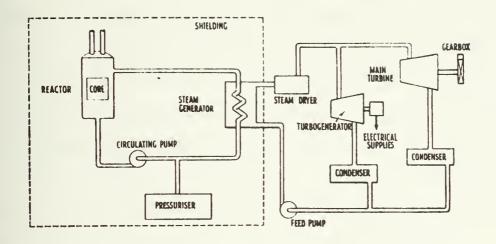


FIGURE 2. BASIC CIRCUIT OF A PRESSURIZED WATER REACTOR (Ref 12)

I.A.3.b. Fuel Cells

Fuel cells convert chemical energy directly into electrical energy, primarily using hydrogen and oxygen as the reactants in a reverse electrolysis process. (See Figure 3)

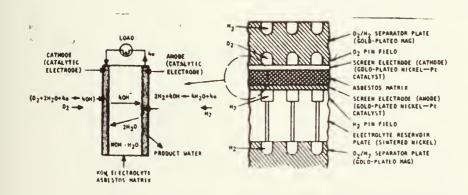


FIGURE 3. H2-O2 FUEL CELL -- CELL DESIGN (Ref 2)



When compared with fuel cells using ammonia, methanol, or hydrocarbons, either directly in the fuel cell or by using reformers to produce hydrogen to use in a hydrogen-oxygen fuel cell, the hydrogen-oxygen fuel cell using hydrogen and oxygen at the reactants offers the best choice. As shown in Table 2, the H₂-O₂ fuel cell has the lowest specific weight and volume, lowest noise output, lowest heat rejection rate, highest thermal efficiency, best consistent response and starting characteristics, and the simplest design.

The hydrogen-oxygen fuel cell has been used in both the APOLLO and GEMINI space programs with great success, and will be used to supply electrical power in the space shuttle. The data on the $\rm H_2-O_2$ fuel cell used in this study was taken from information on the fuel cell designed for use in the space shuttle. It offered the most extensive and up to date data available in the literature. (See Table 3 for some details.)

| | APOLLO PROGRAM | TECHNOLOGY PROGRAM | SHUTTLE PROGRAM |
|-----------------------------|-------------------|-----------------------|------------------------------------|
| LIFE (HOURS | 400 | 5000 | 2000 5000 (WITH MAINTENANCE) |
| SUSTAINED POWER (KW) | 0.9 | 5 | 7 |
| PEAK POWER (KH) | 1.4 | 10 | 12 |
| SPECIFIC WEIGHT (LB/KM) | 170 | 30-40 | 35 |
| SPECIFIC VOLUME (FT3/KW) | 5 B | 0.25 | 0.96 |
| START-STOP CYCLES | 12 | UNLIMITED | 125 |

TABLE 3. A COMPARISON OF FUEL CELL REQUIREMENTS FOR APOLLO, NASA TECHNOLOGY, AND THE SHUTTLE PROGRAM (Ref 2)

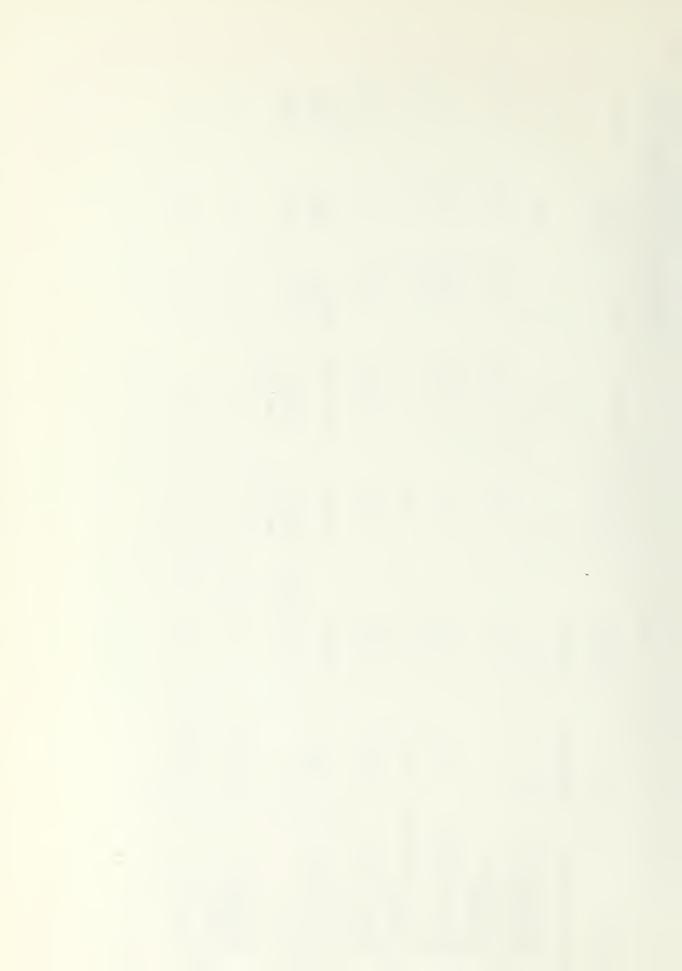
I.A.3.c. Fuel Storage Systems

Hydrogen and oxygen are gases at atmospheric temperatures and pressures, with very low densities. There are three feasible ways of storing hydrogen: as a high pressure

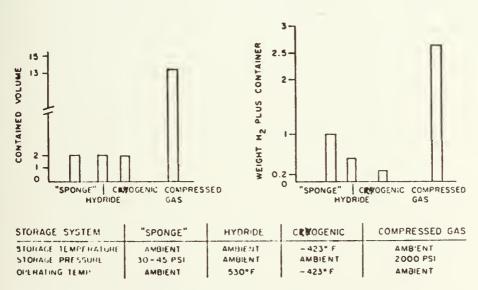


TABLE 2. COMPARISON OF FUEL CELL TYPES (Ref 7)

| INDIRECT - H_2 -0 ₂ fuel cell | Hydro- Methanol carbon | | 65 52 | 1.21 1.00 | 0.93 1.28 | 1.563 2.040 | 6.72 9.80 | 2500- 2500- 4000 4000 | high high | 9 | 57 |
|--|---------------------------|-------------|-----------------------|------------------------------------|------------------------------|--------------------------|----------------------|-------------------------------|-------------------|-----------------------------------|------------|
| INDIRECT | Ammonia | | 89 | 1.32 | 1.06 | 1.886 | 8.99 | 2500- 4000 | high | 4 | 4 |
| | Methanol | Acid | 165 | 1.88 | 1.38 | 2.100 | 10.10 | 500-1000 | inter- mediate | 2 | 2 |
| DIRECT | Hydrocarbon (200°C) | Acid | 125 | 1.64 | 1.33 | 1.800 | 7.68 | 400-800 | inter- mediate | m | т |
| | Ammonia (40°C) | Akaline | 31 | 0.47 | 1.27 | 1.930 | 9.25 | 400-800 | intermediate | 7 | 2 |
| | н ₂ | Akaline | 26 | 0.41 | 0.85 | 0.925 | 9.25 | 2500- 4000 | low | l(best) | l(best) |
| | Fuel | Electrolyte | Weight/power (lbs/KW) | Volume/power (ft ³ /KW) | Heat rejection ratio (KW/KW) | Total reactants (1b/KWH) | Energy costs (¢/KWH) | Projected plant life hours | Noise | Transient Response Start-Up | Simplicity |



gas, a cryogenic liquid or in a metal hydride. Alternative ways of producing hydrogen from hydrocarbons were ruled out based on complexities, further heat rejection requirements, and lack of reliability. Table 2 shows the disadvantages of this method. Figure 4 shows comparitive volumes and weights for hydrogen storage systems. After deciding the submarine could not absorb the penalty of the extra weight of the metal hydride system, the storage system chosen for hydrogen was cryogenics. Cryogenic storage for oxygen was also chosen. The safety problem is probably the greatest with cryogenic storage but with proper design and handling, this system will be no more dangerous than a nuclear reactor.



"SPONGE" SYSTEM . H2 ABSORBED IN LaNIS RARE EARTH COMPLEX
HYDRIDE SYSTEM : MAGNESIUM HYDRIDE

FIGURE 4. COMPARISON OF VARIOUS HYDROGEN STORAGE SYSTEMS (Ref 17)



I.B. Assessment

The outputs of the math model are maximum velocity, payload mass and volume, cruising range, and capital cost. A submarine is considered infeasible when its payload mass or volume is less than zero. The two power plant types will be assessed by two methods: Maximum velocity feasible based on vehicle size, power, endurance, and payload mass fraction, and capital cost based on vehicle size, and power, or based on size and maximum velocity.



CHAPTER II

MATH MODEL

The math model described in this section is based on Ref. 5, Conceptual Design of Submarines (CODESUB). The study is based on data from existing U.S. Navy submarines, both attack type and ballistic missile carrying and is constrained to nuclear propulsion plants, and hulls which are body of revolutions and single-shaft configurations. The displacement ranges are from 2607 to 8251 LTONS. CODESUB takes the NAVSHIPS weight groups and breaks it into finer groups. (See Table 4) This study will use the NAVSHIPS weight groups.

NAVSHIPS Weight Groups

- 1. Hull
- 2. Propulsion
- 3. Electric
- 4. Communication and Control
- 5. Auxiliary Systems
- 6. Outfit and Furnishings
- 7. Armament Loads

Code Sub Weight Groups

- 1. Hull
- 2. Main Propulsion
- 3. Electric
- Communication and 4. Control
- Auxiliary Systems 5.
- 6. Outfit and Furnishings
- 7. Armament
- 8. Battery
- Emergency Propulsion 9. Motor
- Personnel 10.
- Variable Load 11.
- 12. Emergency Fuel
- 13. Water Ballast
- 14. Fixed Ballast
- 15. Special Payload

16. Cargo

NAVSHIPS AND CODESUB WEIGHT GROUPS TABLE 4.



The CODESUB program strives to remove payload items from the NAVSHIPS weight groups. Payload items are divided into three types; electronic equipment normally in NAVSHIPS Group 4, Communication and Control, weapons systems including all expendable items such as torpedoes and mines, (normally NAV-SHIPS Group 7) and cargo, including cargo handling gear. This program is ideally suited to the task set out in this study; calculating weights solely based on displacement and power. Part of the payload in this study will be the personnel other than that required to run the submarine and its propulsion plant.

Unfortunately, CODESUB is based on only a limited number of submarines, and does not include the latest nuclear submarines, the SSN688 LOS ANGELES class or the TRIDENT submarines. Comparisons with data from these classes plus data from a design study on a 60,000 SHP submarine does lend support to the CODESUB program. Maximum errors less than 10%.

The controlling factor in the CODESUB program is weight.

Based on weight, volumes, labor hours and material, costs are

calculated. Weight is the most easily obtained data on sub
marines and therefore is the most accurate.

Cost analysis from CODESUB is in 1971 dollars. Man hours and material costs are converted to 1976 dollars in this study. CODESUB breaks down cost into two groups, labor and material. In addition, the basic CODESUB construction cost differs from the NAVSHIPS basic construction cost by including the cost of



government furnished equipment. The resulting equations in CODESUB sometimes results in large differences from empirical values but the estimated values are considered more realistic for predicting costs for conceptual submarines since market conditions, status of learning, etc., are difficult to forecast.

In order to calculate cost information in 1976 dollars, several changes were made. Due to inflation the cost of material was incremented by 75.4%. Changes in productivity resulted in an increase of 15% in the man-hours. Labor man-hour rate was \$6.84/man-hour and design and engineering labor rate was \$8.79/man-hour. Profit and overhead accounted for another 5% and 8% respectively of the total capital cost.

The center of gravity calculations were followed exactly from CODESUB. The center of gravity for the required payload was calculated to insure the feasibility of the design.

II.A. Size and Power Calculations

Displacement is fixed as an independent input. Using the data listed in CODESUB, a value for the prismatic coefficient, C_p and L/D (length/diameter) ratio was chosen. Based on attack submarines, a value of 0.70 for C_p and a value of 9.0 for L/D were chosen. These values are necessary inputs into powering calculations. (See Table 5 for sizing information) Powering calculations are detailed in Appendix I. The results are based on the equation:

Power =
$$k\Delta^{2/3}$$
 y^3



Appendix I details the selection of k, which is a function of speed, L/D, diameter (calculated from C_p), and an additional roughness allowance, ΔC_f which was assumed to be 0.0004.

TABLE 5
Characteristics of Input Submarines

| (tonnes) | Ср | L/D | L (ft) | D (ft) | Propeller Diameter (ft) |
|----------|------|-----|-----------|-----------|-------------------------------|
| 1000 | 0.70 | 9.0 | 172 | 19.1 | 12.7 |
| 2000 | 0.70 | 9.0 | 216.5 | 24.1 | 16.0 |
| 5000 | 0.70 | 9.0 | 294 | 32.7 | 21.8 |
| 10000 | 0.70 | 9.0 | 370.5 | 41.1 | 27.4 |

II.B. Hull - Group 1

The hull group includes the pressure hull and pressure proof enclosures whose mass is dependent on test depth and yield strength of the hull material, the associated framing and bulkheads, and foundations for equipment included in the other groups. Inputs into the equation are displacement, test depth, (t_D) which is two-thirds of crush depth, yield strength of the steel (σ_y) , density of the steel (ζ_s) , and main propulsion mass (W2).

W1 = $[0.1597 + (3.473 + 0.018 \zeta_s)(t_D/\sigma_y)]\Delta + 0.1451[W2]$ (see Table 6)

$$\zeta_{s} = 495 \text{ lb/ft}^{3}$$
 $t_{D} = 1312 \text{ ft}$ $\sigma_{y} = 80,000 \text{ psi}$



TABLE 6
Group 1 Masses in Tonnes

| SHP | Δ-1000 | 2000 | 5000 | 10000 |
|--------|--------|--------|------|-------|
| 5000 | 421 | 784 | 1872 | 3686 |
| 10000 | 452 | 815 | 1903 | 3717 |
| 20000 | 504 | 866 | 1955 | 3769 |
| 30000 | 549 | 912 | 2000 | 3814 |
| 40000 | 591 | 954 | 2042 | 3856 |
| 50000 | 631 | 994 | 2082 | 3896 |
| 60000 | 669 | 1032 | 2121 | 3935 |
| 80000 | 743 | 1106 | 2194 | 4008 |
| 100000 | 814 | 1177 | 2265 | 4079 |
| | | | | |
| | | FUEL C | ELL | |
| SHP | △-1000 | 2000 | 5000 | 10000 |
| 5000 | 390 | 756 | 1852 | 3678 |
| 30000 | 4.5.4 | | | |

| SHP | △-1000 | 2000 | 5000 | 10000 |
|--------|--------|------|------|-------|
| 5000 | 390 | 756 | 1852 | 3678 |
| 10000 | 414 | 780 | 1876 | 3702 |
| 20000 | 461 | 827 | 1924 | 3750 |
| 30000 | 508 | 874 | 1971 | 3798 |
| 40000 | 556 | 922 | 2019 | 3845 |
| 50000 | 603 | 969 | 2066 | 3893 |
| 60000 | 650 | 1017 | 2114 | 3941 |
| 80000 | 745 | 1111 | 2209 | 4037 |
| 100000 | 840 | 1206 | 2304 | 4132 |

NOTE: Fuel cell masses do not include fuel tank mass which would account for an additional 5-160 tons depending on the displacement and endurance.



The required volume for group 1 is taken as zero. This convention may be followed without discrepancy in the total required volume of the submarine because volumes required by other components are taken to the outside of the hull rather than to interior molded lines.

The center of gravity for hull mass is based on the hull diameter:

$$Z_1 = 0.4953D$$

Initial expection was that the center of gravity would be exactly half of the diameter, but the foundations and other equipment accounts for the difference.

It would be desirable to estimate costs for both labor hours and material for the pressure resistant portion of Group 1 separately from the other structures. This is because this portion requires more careful fabrication technique and the use of higher strength and more costly steels. However, the data breakdown does not permit this. All data used was based on an HY-80 hull material, so correction for different costs due to hull material was not necessary.

The labor hours for construction of the hull may be estimated by:

$$H1 = 13800 (0.9842 \cdot W1)^{0.57}$$



The estimation equation for hull material dollars is:

$$M1 = 220302.4 (0.9842 \cdot W1)^{0.41}$$

II.C. Main Propulsion - Group 2

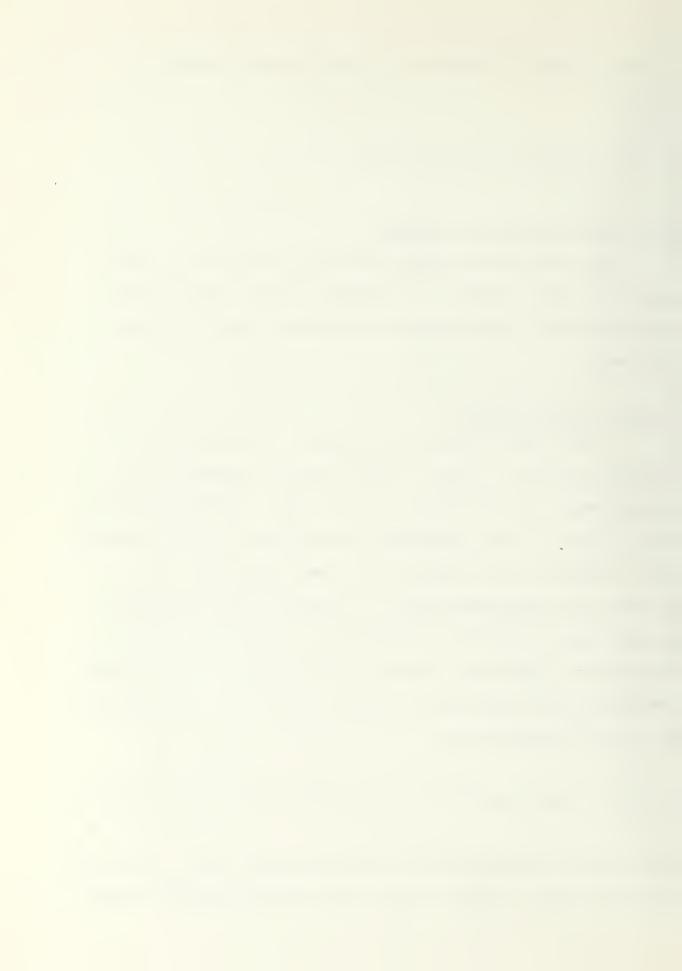
The calculation process for main propulsion for each propulsion plant differs. The nuclear reactor calculations follow CODESUB. The fuel cell calculations are calculated differently.

II.C.1. Nuclear Reactor

This group includes the reactor, reactor vessel, energy converters, a large storage battery, emergency propulsion motor, and a number of other items. CODESUB assumes that the mass of some components varies directly with installed shaft horsepower, and the mass of other components varies directly with the square root of installed shaft horsepower. An additional correction is included to account for sound isolation of machinery. The mass of the storage battery and emergency propulsion motor is taken to be fixed. The equation is of the following form:

$$W2 = 4.462 \text{ (SHP)}^{1/2} + 0.0170 \text{ (SHP)} + 79.96 \text{ (see Table 7)}$$

The volume is calculated from the mass result, and a value for specific volume. Again a fixed value for the battery and EPM



is added. (see Table 7)

$$V2 = [(W2 - 79.96)(48.28) + 5865] 0.02832$$

The center of gravity is estimated from the following:

$$Z2 = 0.4505D$$

The cost of group 2 and group 3 (the electric plant) is calculated together due to their close relationship during construction. As in group 1, data scatter was considerable, and the estimation equations are guess estimates. The labor calculation is:

$$H2 + H3 = 679.1(W2 + W3)$$

The material equation, including the cost of government furnished equipment, is:

$$M2 + M3 = 4488.4(W2 + W3) + 43313 W2$$

II.C.2. Fuel Cell

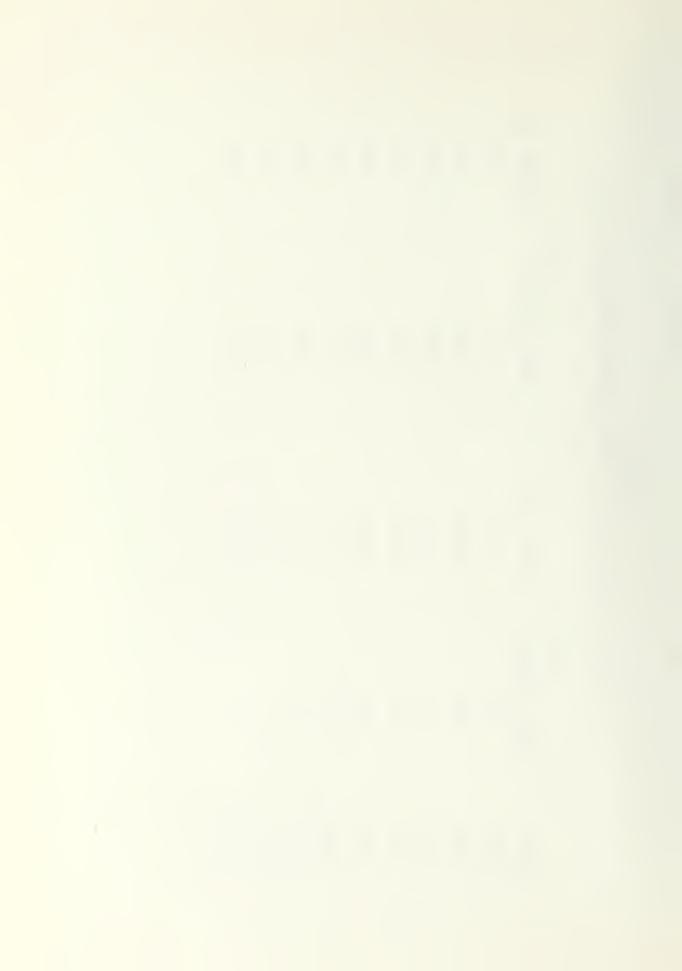
Fuel cell group 2 mass is calculated in four steps; estimating elective load, calculating the fuel cell mass, calculating the D.C. main propulsion motor mass, and calculating shafting, propeller, etc., mass. Based on installed



TABLE 7

Group 2 Masses and Volumes - Nuclear Reactor

| | Group 2 | | Group 3 | ٣ |
|--------|---------------|--------------------------|---------------|--------------------------|
| SHP | Mass (tonnes) | Volume (m ³) | Mass (tonnes) | Volume (m ³) |
| 2000 | 480 | 714 | 72 | 114 |
| 10000 | 969 | 1009 | 104 | 166 |
| 20000 | 1051 | 1494 | 158 | 250 |
| 30000 | 1363 | 1920 | 204 | 324 |
| 40000 | 1652 | 2316 | 248 | 393 |
| 50000 | 1928 | 2692 | 289 | 459 |
| 00009 | 2193 | 3055 | 329 | 522 |
| 00008 | 2702 | 3751 | 405 | 643 |
| 100000 | 3191 | 4420 | 479 | 760 |



shaft horsepower, and vehicle mass, the propulsive kilowatt load is calculated, assuming an efficiency of 90% for the D.C. motor, 10% parasitic loads within the fuel cell. An electric load for support equipment based on vehicle mass is then added (see Table 8). Based on this kilowatt load, a mass is calculated using a specific weight of 35 lb/KW. Propulsion motor mass is calculated using a 36 lb/HP specific weight and allowing a 10% error margin. Assuming a propeller diameter two-thirds the diameter of the ship, a shaft length a maximum of 20% the length of the ship, an equation based on shaft horsepower was used to calculate the remaining mass. Details are in Appendix II.

Volume calculations are also divided into three parts. The actual fuel cell volume is calculated using 0.96 ft³/KW and then corrected by adding a 10% factor for errors. Main propulsion motor volume is calculated by using a specific volume of 0.1 ft³/HP and correcting this figure by adding a 10% error margin also. Volume for the remaining group 2 is calculated using this specific volume from CODESUB and the mass of the remaining group. Masses and volumes are given in Table 9.

The center of gravity is taken to be identical to the nuclear reactor center of gravity:

22 = 0.4505D



TABLE 8
Fuel Cell Non Propulsive Elective Load and Fuel Consumption

| Δ (tonnes) | Maximum Electrical Load (KW) | Fuel Consumption (50% load) | (Kg/Hr) |
|------------|------------------------------------|-----------------------------|---------|
| 1000 | 993 | 180.1 | |
| 2000 | 1717 | 311.5 | |
| 5000 | 3542 | 642.5 | |
| 10000 | 6124 | 1110.9 | |



| SHP | ∧-1000 | 2000 | 5000 | 10000 |
|--------|----------------------------------|-----------|-----------|-----------|
| 5000 | 188 tonnes 181 m ³ | 209/213 | 264/298 | 344/422 |
| 10000 | 351/321 | 372/354 | 428/440 | 509/565 |
| 20000 | 678/603 | 699/636 | 756/723 | 839/851 |
| 30000 | 1004/884 | 1026/918 | 1083/1007 | 1168/1138 |
| 40000 | 1330/1166 | 1352/1200 | 1411/1290 | 1498/1424 |
| 50000 | 1656/1447 | 1679/1483 | 1738/1574 | 1828/1710 |
| 60000 | 1982/1729 | 2006/1765 | 2066/1857 | 2157/1996 |
| 80000 | 2635/2291 | 2659/2329 | 2721/2424 | 2817/2568 |
| 100000 | 3287/2854 | 3313/2894 | 3377/2991 | 3476/3141 |



The amount of labor required for construction of group 2 for fuel cells was assumed to be identical to nuclear power.

H2 = 679.1 W2

The material costs for the fuel cell was also divided into three calculations. Due to the low level of mass produced fuel cells, the majority being prototypes designed for unique uses, an accurate cost figure on fuel cells is not available. Ref. 6 estimates the capital cost at \$120/KW for a land-based, commercial power plant. Due to unique quality controls required by the Navy, plus other requirements based on shock, transients motions, and other military specifications, the cost to the Navy is probably higher. Not including the research and development cost, the cost used in this study was \$500/KW. Costs for direct current propulsion motors were calculated using \$100/HP. The remaining material cost of constructing group 2 was used using the cost data from CODESUB and the remaining group 2 weight. Labor hours were calculated using the total group 2 mass and the CODESUB labor figures from Group 2.

Details of fuel cell group 2 shafting and propeller mass calculations are given in Appendix II.



II.D. Electric Plant - Group 3

II.D.l. Nuclear Reactor

generation equipment and the power distribution equipment.

CODESUB uses vehicle mass to calculate this mass. While this may be correct for the limited number of power outputs used in CODESUB, for this study it is not adequate. Coolant pumps circulating water through the reactor core and steam generators constitute a major electric load. This load is a function of horsepower. Using both CODESUB and other data available in classified sources, electric plant weight was calculated using group 2 mass as an input.

W3 = 0.15 W2

This equation allows for an electric load of 25% of the installed shaft horsepower.

CODESUB is used to calculate the volume:

V3 = W3 (1.587) (see Table 7 for Group 3 results)

The vertical center of gravity is a function of diameter:

23 = 0.5371 D



The labor hours and material costs were calculated along with group 2 as the two groups are closely related during construction.

II.D.2. Fuel Cell

As the fuel cell generates direct current electricity, there is little need for energy converters. A majority of pumps which comprise a major electric load may be D.C. driven. A small capability for providing A.C. is necessary for controls and other normally A.C. available equipment. The electric plant mass is assumed to be both a function of shaft horsepower and displacement and is calculated in the following equation.

$$W3 = \frac{(SHP)}{10,000} + \frac{(\Delta)^{1/2}}{10}$$

The remainder of the fuel cells group 3 calculations are identical to the nuclear reactor case, using the calculated group 3 mass as an input. (see Table 10)

II.E. Communication and Control - Group 4

Group 4 calculations for both power plants are assumed to be identical. The majority of the mass in this group on a normal submarine is accounted for by the payload group. In this study this mass represents a small percentage of both the actual submarine mass and the total NAVSHIPS group 4 mass. Although CODESUB does make numerical estimates and derives



TABLE 10

Fuel Cell Group 3 Masses (TONNES) and Volumes (m^3)

| 10000 | 11/17 | 11/17 | 12/19 | 13/21 | 14/22 | 15/24 | 16/25 | 18/29 | 20/32 |
|---------|---------------------------|-------|-------|-------|-------|-------|-------|-------|--------|
| 2000 | 8/12 | 8/13 | 9/14 | 10/16 | 11/18 | 12/19 | 13/21 | 15/24 | 17/27 |
| 2000 | 5/8 | 5/9 | 6/10 | 7/12 | 8/13 | 9/15 | 10/17 | 12/20 | 14/23 |
| △ -1000 | 4 TONNES/6 m ³ | 4/7 | 5/8 | 6/10 | 7/11 | 8/13 | 9/15 | 11/18 | 13/21 |
| SHP | 2000 | 10000 | 20000 | 30000 | 40000 | 20000 | 00009 | 80000 | 100000 |



a percentage of displacement for an answer, the reader is reminded that these are estimates and are subject to the judgement of the estimator. The displacement dependent part of group 4 mass is calculated from:

$$W4 = 0.0066\Delta$$

The volume of group 4 is calculated using the specific volume given in CODESUB:

$$V4 = W4 \cdot 1.5168$$

The center of gravity is a function of diameter,

$$24 = 0.75D$$

The labor hours and material costs for the displacement dependent part of group 4 are given as follows:

$$H4 = ((W4 \cdot (0.9842)^{0.76}))$$
 5980 labor hours

II.F. Auxiliary Systems - Group 5

Fuel cell group 5 mass was assumed to be less than the nuclear power mass for several reasons: 1) lower heat output at lower temperatures requiring a smaller airconditioning



and sea water cooling load, 2) smaller refrigeration and air conditioning loads due to smaller requirements for personnel, and 3) smaller life support requirements due to smaller requirements for personnel plus the elimination of either oxygen storage or generation for metabolic reasons by simply adding to fuel storage requirements. For the above reasons, the fuel cell group 5 mass was assumed to be 85% of the nuclear power group 5 mass, and calculations using this mass are also correspondingly less.

The following equations are used to estimate the necessary characteristics of group 5.

| | | NUCI | LEAR | | | | FUEL C | ELL | | |
|------------|---|-----------|------------------------|--------------------|-----|---|--------|----------|----|-------------------|
| W5 | = | DISPL | (0.0771) | | W5 | = | DISPL | (0.0665) | | mass |
| V5 | = | W5 · (1. | . 238) | | | | same | | | volume |
| Z 5 | = | D(0.53 | 379) | | | | same | | of | center gravity |
| Н5 | = | ((W5 · (| 0.9842) ^{0.5} | ²)3013 | 30 | | same | | | labor hours |
| М5 | = | ((W5• (| 0.9842) 0.5 | ⁹)1613 | 368 | | same | | | material cost |

II.G. Outfit and Furnishings - Group 6 and Personnel

Group 6 calculations are functions of displacement, number of personnel, and endurance. This group is the combination of CODESUB's group 6 and 10. Endurance for group 6 calculations was fixed at 60 days. The difference between calculations for endurance power and fuel cells exists in the number of personnel required to man the necessary watch



stations at sea, and perform required maintenance. Nuclear reactors require a large number of personnel to run, while fuel cells require only a minimum number. Estimation of fuel cell manning is only a rough estimate. The fuel cell can be easy to run, based on existing smaller power plants. Table 11 gives manning levels for each power plant.

TABLE 11

Manning Levels for Submarine Power Plants

| | Nuc | lear | Power | | Fuel Cell |
|-------|---------------|-----------|----------------|-------------|---|
| SHP O | officer N3 | CPO N2 | Enlisted Nl | Total N4 | Officer CPO Enlisted Total N3 N2 N1 N4 |
| 5000 | 3 | 3 | 23 | 29 | 2 2 11 15 |
| 10000 | 3 | 3 | 26 | 32 | 2 2 12 16 |
| 20000 |) 4 | 4 | 33 | 41 | 3 3 14 20 |
| 30000 |) 4 | 4 | 40 | 48 | 3 3 16 22 |
| 40000 |) 4 | 5 | 46 | 55 | 3 4 18 25 |
| 50000 |) 4 | 5 | 53 | 62 | 3 4 20 27 |
| 60000 |) 4 | 6 | 60 | 70 | 3 5 22 30 |
| 80000 |) 4 | 7 | 73 | 84 | 4 6 26 36 |
| 10000 | 00 5 | 8 | 86 | 99 | 4 7 30 41 |

Based on the above inputs, the following equations are used to estimate group 6 characteristics:

 $W + W + W \leftarrow (400 (N3) + 330 (N2) + 230 (N1) + 420.6 (N4)]/2205 + \Delta (0.02884)$



$$V_p + V_{pr} + V6 = (N4)(7.557)$$

$$z_p + z_{pr} + z_6 = D \cdot 0.5422$$

$$H6 = 63.02\Delta$$

$$M6 = 230475.6 (0.02838\Delta)^{0.37}$$

II.H. Armament - Group 7

Armament is classified as payload so all characteristics are zero.

II.I. Ballast

CODESUB classified variable loads as group 11 and water ballast as group 13 and fixed ballast (lead) as group 14.

Lead is used both for a future growth margin and for stability. The minimum amount of lead used was assumed to be 4% of the displacement. Water ballast is limited to a minimum of 12% of the displacement in order to insure adequate reserve bouyancy on the surface. Variable loads, non-ballast water, consumable lube oil, compressed air, account for about 1.63% of the displacement. Incorporating these three groups into one, the following equations are used to obtain estimates of ballast characteristics:



 $W7 = 0.1763 \Delta$

V7 = 0.7759 W7

Z7 = 0.3716 D

 $M7 = 29.53 \Delta$

II.J. Fuel, Storage Tanks, and Endurance

Nuclear power reactors consume uranium fuel but this amount is negible. The amount of fuel required by nuclear power is assumed to be zero, and the endurance is only limited by personnel considerations, which limited mission time to sixty days. Endurance range is based on the speed maintained for sixty days.

Calculations for fuel cell requirements were much more difficult. Based on half of the non-propulsive load (Table 8), and endurance speed (10 kts was chosen), an electrical load on the fuel cell was estimated. Based on this load and the specific reactant consumption for the fuel cell (0.8 lb/KWH), the masses of the reactants were obtained. Using the ratio of 7.94 to 1 for the mass of oxygen to mass of hydrogen, and their respective densities $(O_2(\ell)-71.4 \text{ lb/ft}^3)$, $(H_2(\ell)-4.43 \text{ lb/ft}^3)$, the volume of reactants was estimated.



The storage tank and insulation masses were estimated to be 20% of the reactant mass. Its volume was estimated using a specific volume of $0.3687 \text{ m}^3/\text{TONNES}$ (density of 169.3 lb/ft^3).

II.K. Summary of Math Model

Table 12 summarizes the equations given in Sections II.B to II.J. The input value of vehicle mass minus the sum of masses included in Table 12 equals vehicle mass available for payload and fuel. Similarly the input value of total vehicle volume minus the sum of the volumes included in Table 12 equals vehicle volume available for payload and fuel. For any given vehicle, then, if endurance is fixed, the vehicle mass and volume available for payload are obtainable. If payload mass is fixed, both endurance and volume available for payload are obtainable. Data is given in Chapter III exploring both of these possibilities.



TABLE 12

Summary of Group Characteristics

Nuclear Sub Fuel Cell Sub HULL $\overline{W1} = 0.3628\Delta + 0.1451 W2$ V1 = 021 = 0.4953 Dsame $H1 = 13800 (0.9842 W1)^{0.57}$ $M1 = 220302.4 (0.9842 W1)^{0.41}$ MAIN PROPULSION $W2 = 4.462 \text{ (SHP)}^{1/2} + 0.0170 \text{ SHP} + 80$ see Table 9 $\zeta_2 = 731 \text{ kg/m}^3 *$ Z2 = 0.4505 Dsame H2 = 679.1 W2same M2 = 47801.4 W2(see Section II.C.2) ELECTRICAL PLANT W3 = 0.15 W2 $W3 = SHP/10000 + \sqrt{\Delta}/10$ $\zeta_3 = 630 \text{ kg/m}^3$ same $z_3 = 0.5371 D$ same H3 = 679.1 W3same M3 = 4488.4 W3same COMM & CONTROL $W4 = 0.0066 \Delta$ $c_{\Delta} = 659 \text{ kg/m}^3$ 24 = 0.75 Dsame $H4 = 5980(0.9842 \text{ W4})^{0.76}$ M4 = 10012.5 W4

*This density applies only to first two terms of W2 equation.



TABLE 12 (cont)

 $M7 = 29.53 \Lambda$



CHAPTER III

RESULTS

III.A. Nuclear Power

The mass fractions and volume fractions for each group for discrete shaft horsepower and vehicle masses are given in Table 13. Characteristics for a test depth of 400 meters (1212 ft) and 200 meters (606 ft) were included to allow the reader to judge the effect on payload the operating depth had. Obviously any total mass or volume fraction greater than one represents an infeasible submarine.

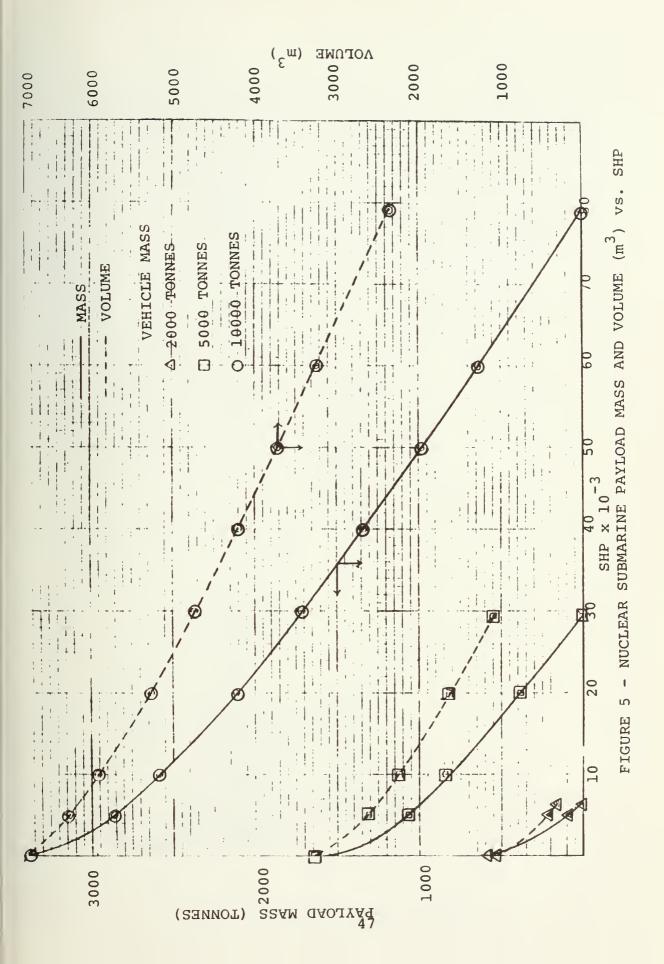
Figure 5 is a plot of payload mass and volume versus shaft horsepower. Figure 5 shows that a 2000 tonnes nuclear submarine is infeasible with more than 6500 H.P. (24 kts), a 5000 tonnes submarine with more than 29,500 H.P. (34 kts) and a 10000 tonnes submarine with more than 79,000 H.P. (41 kts). At these limits the submarines are mass limited. The volume limit on feasible vehicles would extend to higher power outputs. Figure 6 plots the density of the payload versus shaft horsepower. By knowing the density of the desired payload a design can be determined to be either mass or volume limited by using Figure 6. If the desired payload item has a lower density than shown in Figure 6, the design will be volume limited. Inputting shaft horsepower and vehicle size does constrain the payload density.

No nuclear submarine was infeasible based on vertical center of gravity requirements.

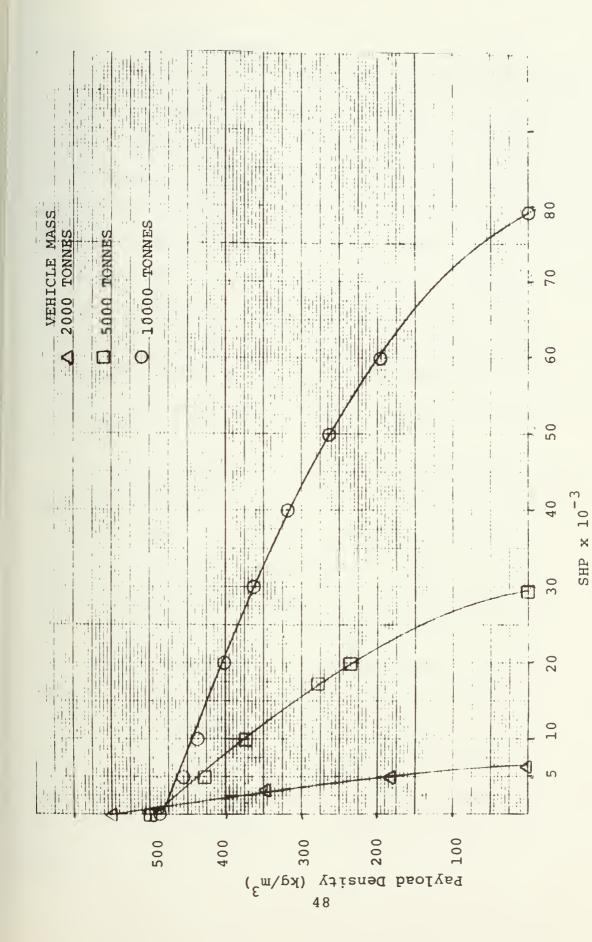


AND VOLUME FRACTIONS GROUP MASS NUCLEAR SUBMARINE









NUCLEAR SUBMARINE PAYLOAD DENSITY VS. SHP, ł 9 FIGURE



III.B. Fuel Cell Submarine

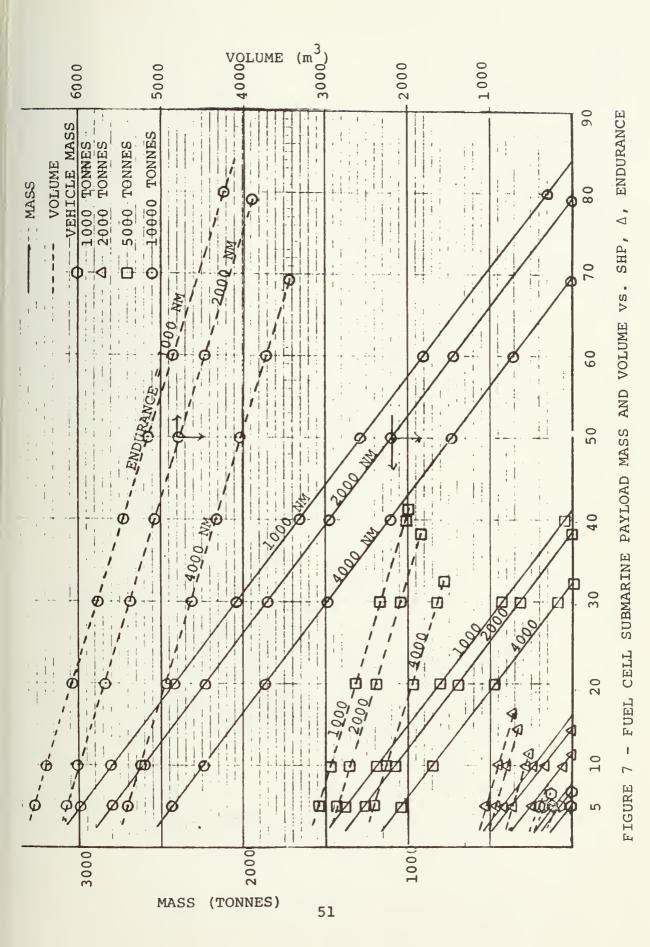
The results for the fuel cell submarine are presented in the same format as the nuclear submarine. The mass fractions and volume fractions are broken down by groups for each input submarine. (see Table 14) As with the nuclear submarine, a total mass fraction or volume fraction greater than one represents an infeasible design. Note that Table 14 does not include fuel and storage tank fraction. Without a specification of endurance, these items cannot be included. Therefore some of the remaining mass and volume would have to include this weight, at the cost of losing some payload. Again the effect of halving the operating depth is demonstrated in Table 14.

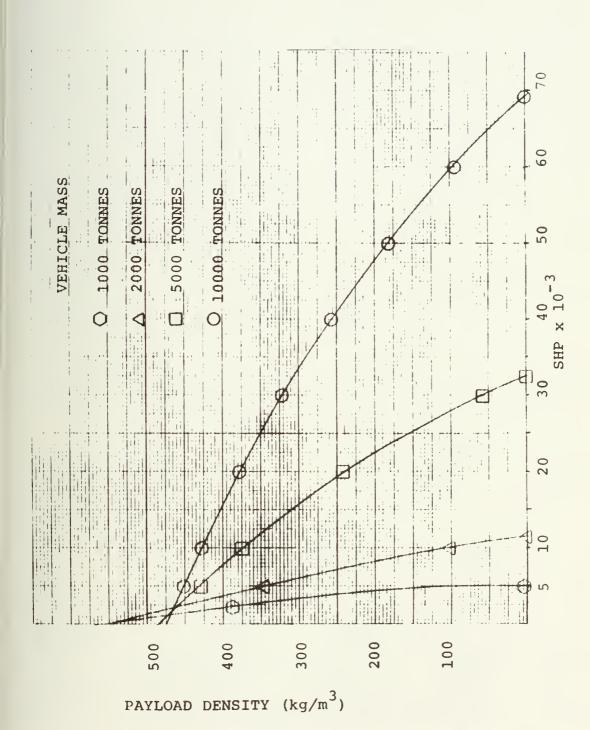
Figure 7 is a plot of payload mass and volume versus shaft horsepower at discrete vehicle masses and endurances. Since endurance is specified in Figures 7 through 12 and in Table 16, the payload mass and volume given in those figures already have fuel and storage tank mass and volume subtracted out. Figure 8 shows that a 1000 tonnes fuel cell submarine with 4000 NM endurance is infeasible at more than 5010 H.P. (26 kts), a 2000 tonnes submarine with the same endurance at 11,500 H.P. (30 kts), a 5000 tonnes submarine at 32,000 H.P. (35 kts) and a 10000 tonnes submarine at 69,000 H.P. (39 kts). Using the approach described for Figure 6, Figure 8 may be used to determine the feasibility of payload for fuel cell submarines.



AND VOLUME MASS SUBMARINES FUEL TABLE







52



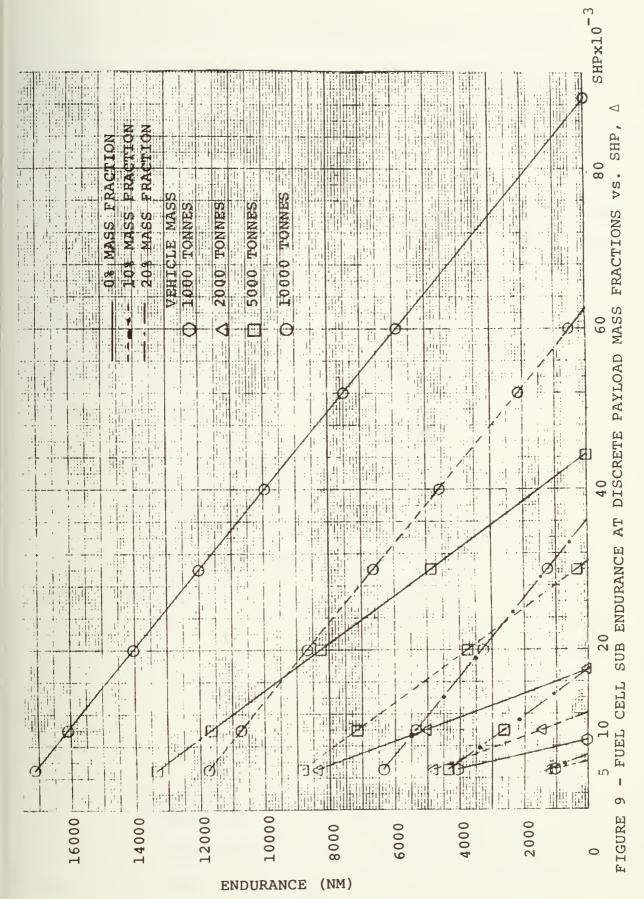




Figure 9 is added to demonstrate the effect of shaft horsepower, displacement and payload mass fraction on the endurance of a fuel cell powered submarine.

No submarine was infeasible based on vertical center of gravity requirements.

III.C. Comparison of Nuclear Reactor and Fuel Cell Submarines

Table 15 lists the maximum velocity for a given shaft horsepower and size. Maximum velocity is not a function of power plant type. In addition, Table 15 lists the cost data for each power plant. The lack of an entry represents an infeasible design. This data is incorporated into figures in the assessment section which follows.

III.D. Results of Assessment Methods

III.D.l. Maximum Velocity Assessment

Figures 10, 11, and 12 are the results of the assessment method which looked at maximum feasible velocity based on vehicle size, power, endurance, and payload mass fraction. The fuel cell submarine data was from calculations based on 2000 NM and 4000 NM endurance ranges. These figures demonstrated the maximum velocity each particular submarine can achieve.

Table 16 summarizes the results from Figures 10-12. It is evident from Table 16 that at small submarine sizes, the fuel cell submarine can achieve higher speeds than a nuclear submarine at respectable values of endurance and carrying the same payload mass as the nuclear submarine. An example of

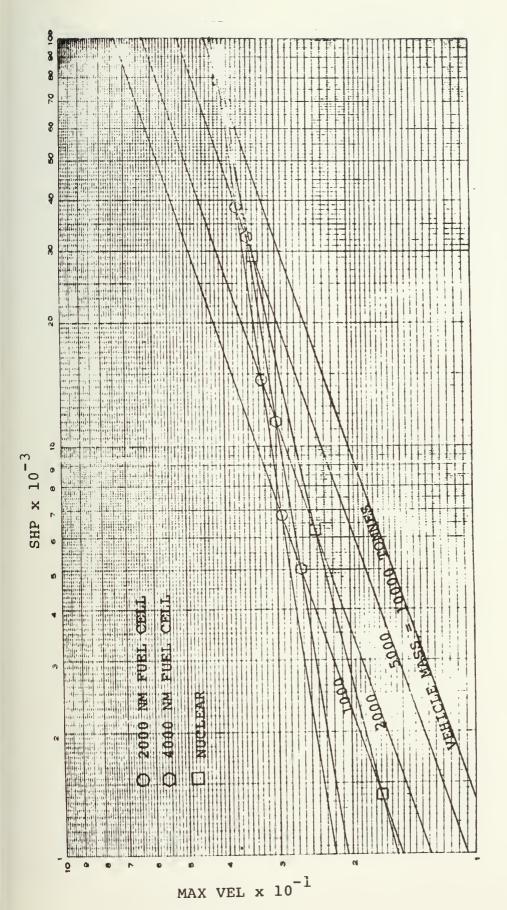


Maximum Velocities and Capital Costs for the Nuclear Submarine and Fuel Cell Submarine with 4000 NM Endurance and No Payload

TABLE 15

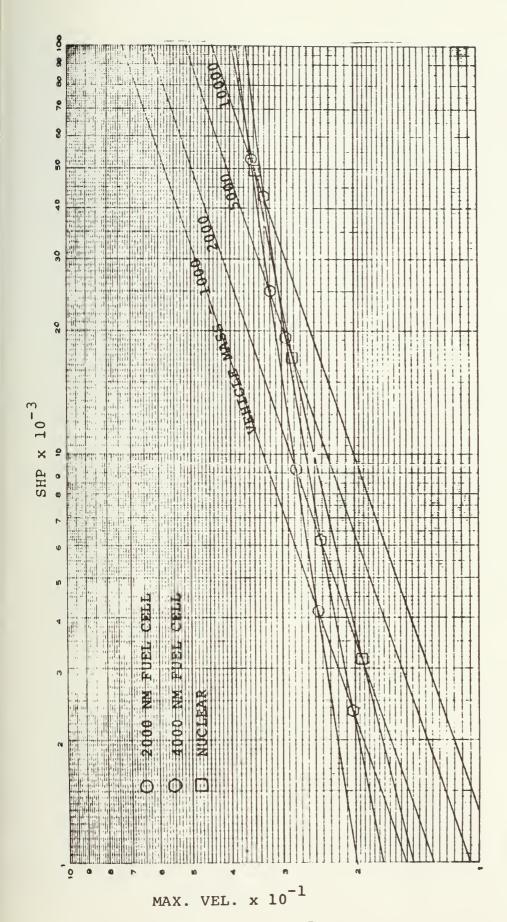
| Vehicle Mass (tonnes) | Shaft Horse- power (HP) | Max. Vel. (knots) | Nuc. Cost (1976\$x10 ⁻⁶) | Fuel Cell Cost (1976\$x10 ⁻⁶) |
|-----------------------------|----------------------------------|----------------------|---|---|
| 1000 | 5000 | 26.2 | | 25.1 |
| 2000 | 5000 | 22.4 | 54.3 | 34.0 |
| 2000 | 10000 | 28.5 | | 39.5 |
| 5000 | 5000 | 18.4 | 71.2 | 53.4 |
| 5000 | 10000 | 23.3 | 84.3 | 58.8 |
| 5000 | 20000 | 29.5 | 105.9 | 69.6 |
| 5000 | 30000 | 34.0 | | 80.4 |
| 10000 | 5000 | 15.8 | 92.4 | 77.7 |
| 10000 | 10000 | 20.0 | 105.4 | 83.1 |
| 10000 | 20000 | 25.4 | 127.0 | 93.8 |
| 10000 | 30000 | 29.2 | 145.9 | 104.6 |
| 10000 | 40000 | 32.2 | 163.5 | 1-15.4 |
| 10000 | 50000 | 34.8 | 180.1 | 126.2 |
| 10000 | 60000 | 37.0 | 196.2 | 136.9 |





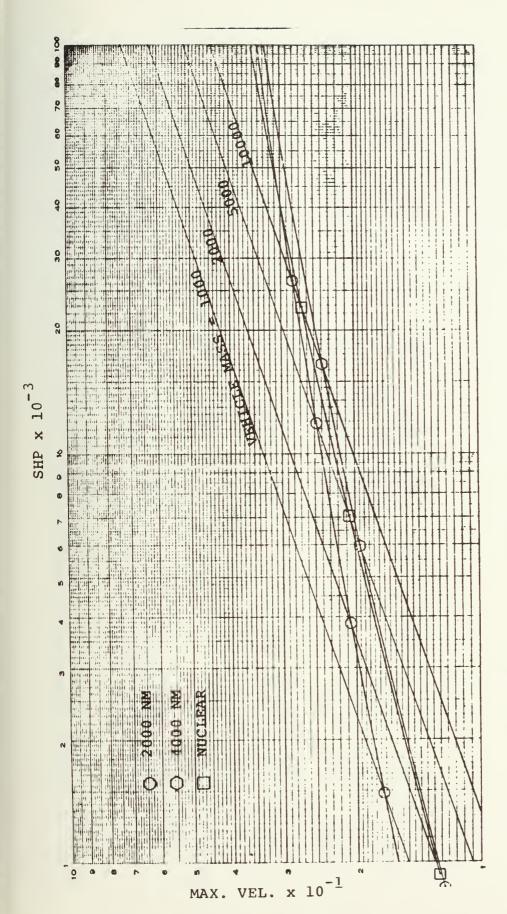
FEASIBILITY LIMIT FOR 0% PAYLOAD MASS FRACTION: MAX. VELOCITY VS. SHP, 2000 NM, 4000 NM FUEL CELL SUBMARINE AND NUCLEAR SUBMARINE FOR Ø ı FIGURE 10





FEASIBILITY LIMIT FOR 10% PAYLOAD MASS FRACTION: MAX. VELOCITY VS. SHP, A FOR 2000 NM, 4000 NM FUEL CELL SUBMARINE, AND NUCLEAR SUBMARINE 1 FIGURE 11



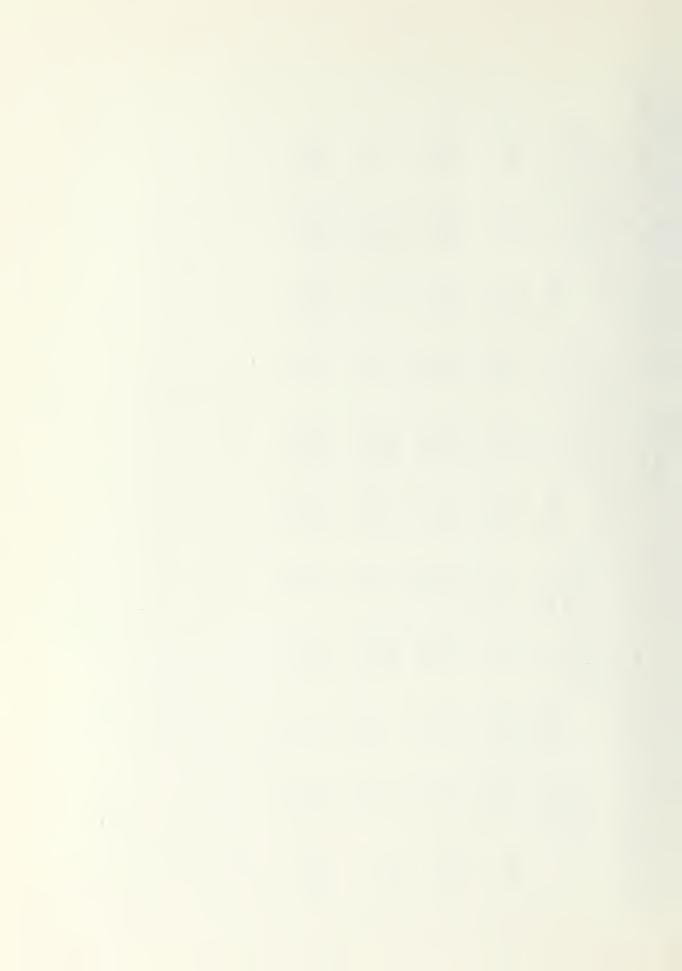


MAX. VELOCITY VS. SHP, A NUCLEAR SUBMARINE SUBMARINE, AND FEASIBILITY LIMIT FOR 20% PAYLOAD MASS FRACTION: NM FUEL CELL 2000 NM, 4000 FOR A ı 12 FIGURE



TABLE 16 - MAXIMUM FEASIBLE VELOCITIES OF SUBMARINES, AND PAYLOAD VOLUME AND DENSITY AS A FUNCTION OF VEHICLE MASS, PAYLOAD MASS FRACTION, POWER PLANT TYPE AND ENDURANCE

| | PAYLOAD MASS | Y Y TAX | NUCLEAR PAYLOAD Y VOLUME | PAYLOAD DENSITY | 2000 NM FUEL PAYLOAD YMAX VOLUME | | CELL PAYLOAD DENSITY | 4000 F | 4000 NM FUEL PAYLOAD P VOLYME D | PAYLOAD DENSITY |
|-------|-----------------|---------|--------------------------------|--------------------|-----------------------------------|--------|----------------------------|---------|---------------------------------|-----------------|
| | FRACTION | (Knots) | | (Kg/m) | (Knots) | | | (knots) | | (Kg/m) |
| 1000 | 0.0 | 17.0 | 44.3 | 0 | 9 | 59. | 0 | 26.2 | 174.2 | 0 |
| | 0.1 | 1 | 1 1 1 1 | | 24.6 | 342.2 | 292 | 0 | 56. | 389 |
| | 0.2 | | 1 | ! | 7 . | 17. | 7 | | | - |
| 2000 | 0.0 | 24.2 | 17. | 0 | 32.3 | 54. | 0 | 29.9 | - | 0 |
| | 0.1 | 19.2 | 579.1 | 346 | 27.7 | 812.8 | 246 | 24.2 | 670.3 | 298 |
| | 0.2 | 12.6 | 41. | 475 | 20.5 | 71. | \vdash | 12.3 | \sim | 483 |
| 5000 | 0.0 | 33.7 | . 66 | 0 | 37.0 | 847. | 0 | 4. | 574. | 0 |
| - | 0.1 | 28.1 | 1790.8 | 279 | 31.9 | 2241.2 | 223 | 29.5 | 1989.2 | 251 |
| | 0.2 | 20.7 | 61. | 406 | 4. | 929 | 376 | 6 | 382. | 420 |
| 10000 | 0.0 | 40.7 | 355. | 0 | 40.7 | 892 | 0 | 38.9 | 437. | 0 |
| | 0.1 | 34.6 | 3745.9 | 267 | 35.4 | 4696.7 | 213 | 33.0 | 4241.2 | 236 |
| | 0.2 | 26.7 | 114. | 391 | 27.9 | 500 | 363 | 23.8 | 090 | 9 |



this is the 2000 tonnes submarine with a payload mass of 200 tonnes, (10% payload mass fraction). The maximum velocity of this submarine with a nuclear plant is 19.3 kts whereas with a fuel cell, the submarine can achieve 27.7 kts at 2000 NM endurance and 24.2 kts at 4000 NM endurance. Referring to Table 16, the latter submarine has volume available for payload equal to 670 m³ and the nuclear submarine has volume for payload equal to 579 m³. Referring ahead to Figure 14, the 2000 tonnes nuclear submarine should cost on the order of \$48,500,000 in 1976 dollars (without payload) whereas the 2000 tonnes, 4000 NM fuel cell submarine should cost on the order of \$35,000,000. Clearly the 4000 NM fuel cell submarine is an attractive competitor to the nuclear submarine in the 2000 tonnes size. Its advantages are that it is five knots faster, has greater payload volume and costs 38% less. Its disadvantage is that it has 4000 NM endurance instead of 60 days x 19.3 knots x 24 hrs/day = 27,500 NM.

On the other hand, at 5000 tonnes some of the advantages of the fuel cell submarine fade. With a payload mass equal to 500 tonnes (10% payload mass fraction), it has only a one knot advantage over the nuclear submarine, its payload volume is constrained to 1989 m³ and compared to 1791 m³ for the nuclear submarine, and its cost without payload is about \$69,000,000 as compared to about \$100,000,000 for the nuclear submarine. At 10000 tonnes it is two knots slower than the nuclear submarine for the payload mass of 1000 tonnes, (payload mass fraction 10%), its payload volume is constrained to less



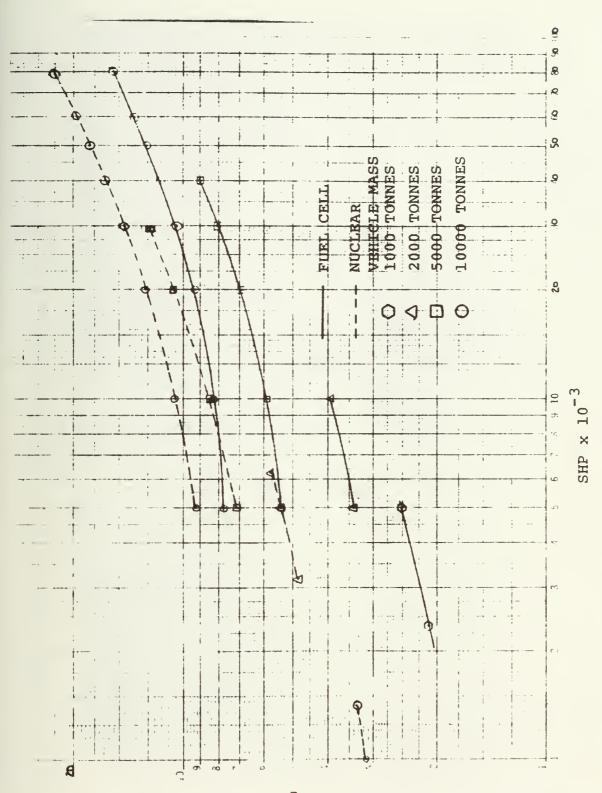
than 4241 m³ as compared to 3746 m³ for the nuclear submarine and its cost without payload is \$119,000,000 compared to \$179,000,000 for the nuclear submarine. Clearly the case for fuel cell submarines is less strong at large sizes than at small sizes as controlled by the constraints of this study.

III.D.2. Capital Cost Assessment

Figure 13 plots cost versus shaft horsepower for discrete vehicle masses. Figure 14 plots maximum velocity versus cost. In both figures the fuel cell submarine shows a clear cost advantage. Data was added for smaller submarines with power outputs below 5000 H.P. to give a clearer picture in the lower power region.

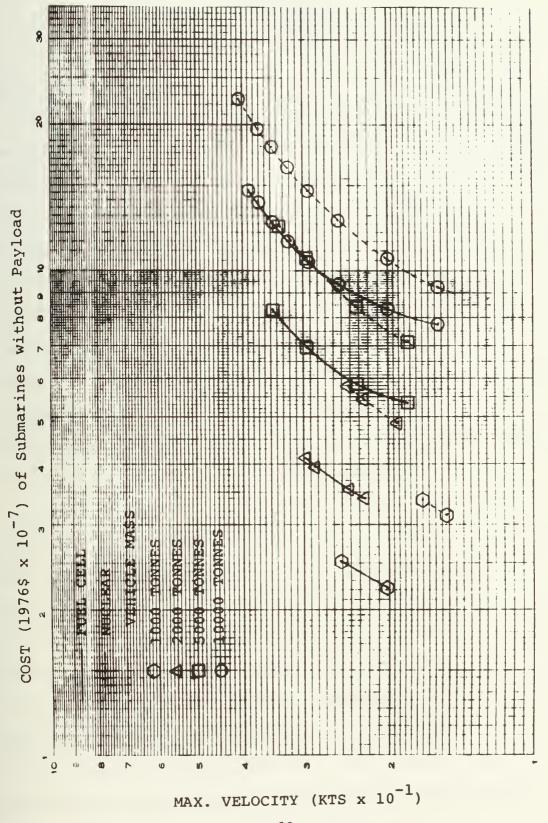
It is evident from Figure 14 that at the \$70,000,000 cost of a 5000 tonnes, 29 knot, 4000 NM endurance, and 500 tonnes (1950 m³) payload fuel cell submarine (not including payload costs), a nuclear submarine of the same size is feasible that has 5000 installed horsepower instead of 20,000 with a speed of 17.5 knots instead of 29 kts. According to Figure 5, this submarine could carry 1122 tonnes of payload with 2619 m³ available for payload volume. Clearly this submarine would be much more attractive for limited speed missions than the fuel cell submarine.





COST (1976\$ x 10^{-7}) of Submarine without Payload





- CAPITAL COSTS FOR NUCLEAR SUBMARINE AND 4000 NM ENDURANCE FUEL CELL SUBMARINE VS. MAX. VELOCITY 14 FIGURE



CHAPTER IV

DISCUSSION

IV.A. Nuclear Reactor

U.S. Navy nuclear powered submarines are divided into two groups, attack submarines and ballistic missile carrying submarines. Traditionally, attack submarines have carried about a 10% payload mass fraction and ballistic missile submarines about a 17-20% payload mass fraction. Therefore the feasibility limits in Figures 11 and 12 are close to actual submarines. The limits for a 0% payload mass fraction are not precise due to the requirement for personnel to drive the submarine and the sensors needed to do that. These items are considered part of the payload. Figure 11 can be used to predict the necessary shaft horsepower and/or displacement to drive a submarine at a required velocity.

Table 13 gives evidence of the penalty one pays for increasing the maximum operating depth. The effect of decreasing the operating depth by 1/2 from 400 meters to 200 meters reduced the hull mass fraction by approximately 10%. This margin can be directly taken up by the payload. Note however, that the volume margins did not change, based on the convention used in this study. Therefore, the density of the payload would increase by decreasing depth, having the effect of raising the lines in Figure 8. As a result, for a given payload and vehicle mass, a submarine will become weight



limited at a greater shaft horsepower. As a result, as the maximum operating depth increases, a submarine increases its chance of being weight limited.

IV.B. Fuel Cell

The fuel cell submarine had one additional parameter affecting its design, namely endurance. The effect of endurance on the payload fractions and the effect of payload mass fractions on endurance are shown in Figures 7 and 9 respectively. Both of these figures demonstrate the penalty endurance inflicts on the payload. Figure 8 is plotted using an endurance of 4000 NM. Lowering the endurance would move the curves in this figure to the right, thereby lowering the shaft horsepower at which the submarine will shift from being volume limited to mass limited for a given payload.

For the same reasons mentioned in the previous section, the 10% and 20% payload mass fractions are the realistic numbers. The fuel cell submarine has identical constraints as the nuclear powered submarine. Changing the maximum operating depth of the fuel cell submarine has the same effect as it did on the nuclear submarine.

IV.C. Comparison of Nuclear Reactor and Fuel Cell Submarines

The differences between payload carrying capabilities of the nuclear powered submarine and fuel cell submarine are small in the middle region of this study. Fuel cell submarines have greater payload carrying capabilities at smaller vehicle



masses. Nuclear powered submarines are infeasible at the lowest vehicle mass, due to the high penalty paid for the reactor plant and the necessary shielding. This penalty is relatively less at larger vehicle masses.

On the other end of the scale, nuclear powered reactors offer similar or better payload carrying capabilities at higher shaft horsepower and vehicle masses. The fuel cell main propulsion mass is based on a linear approximation, while the nuclear reactor is not. Therefore the fuel cell does not show any improvement in specific weight at large power outputs. Realistically, the fuel cell should show some improvement with increasing power outputs but just how much is not known. The specific weight was fixed at 35 lb/KW rather than introduce an arbitrary number. If the fuel cell has a greater specific weight at larger power outputs, the payload carrying advantage evident at low shaft horsepower and vehicle masses would also be maintained throughout the input range.

The fuel cell submarine has a clear advantage over the nuclear powered submarine based on cost. Figures 13 and 14 show that the fuel cell submarine is more than 20% cheaper throughout the input range. As stated in the Math Model Section, the cost for the fuel cell was estimated at \$500/KW. The effect of doubling this cost would be to raise the cost of the fuel cell submarine up to the level of the nuclear submarine, and even surpass it at high shaft horsepower.



CHAPTER V

CONCLUSIONS AND RECOMMENDATIONS

Within the constraints and parameters used in this study the fuel cell powered submarine offers better or equal payload carrying capabilities at a substantially lower cost. The nuclear powered submarine has the advantage of a much higher endurance limit, only limited by personnel requirements. The U.S. Navy has not changed the basic design of the pressurized water nuclear reactor for twenty years.

The fuel cell powered submarine will require substantial studying before it can be built. A specific fuel cell would have to be designed. This fuel cell would be many times larger than the one designed for the Space Shuttle. By taking into account its intended use, specific characteristics would change. Possibly the specific weight and volume will be lower. Its life would definitely have to be improved before it will be used in submarines. The propulsion motor offers another chance for improvements. Superconducting motors have been designed which will lower the weight and volume of the motor and improve its efficiency. The improvement in efficiency alone should extend endurance ranges of the fuel cell submarine another 5%. The fuel storage problem will have to be reevaluated for the specific design. Metal hydride storage of hydrogen may prove to be safer and better than cryogenic storage. Personnel will have to be trained plus repair and maintenance facilities have to be designed. Given



a similar effort as the nuclear power program received in the 1950's, the fuel cell powered submarine could replace nuclear powered submarines in some mission areas.

The fuel cell powered submarine can never totally replace nuclear powered submarines. There are many missions where endurances at high speeds are necessary. However, there are some missions where the fuel cell powered submarine can do the job at much lower costs.



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APPENDIX I

RESISTANCE AND POWERING CALCULATIONS (Ref 10)

SHP =
$$k \Delta^{2/3} (y)^3$$

△ in LTONS

y in knots

INPUTS L/D = 9

$$\Delta C_f = 0.0004$$

D = 19.1 ft., 21.1, 32.7, 41.1

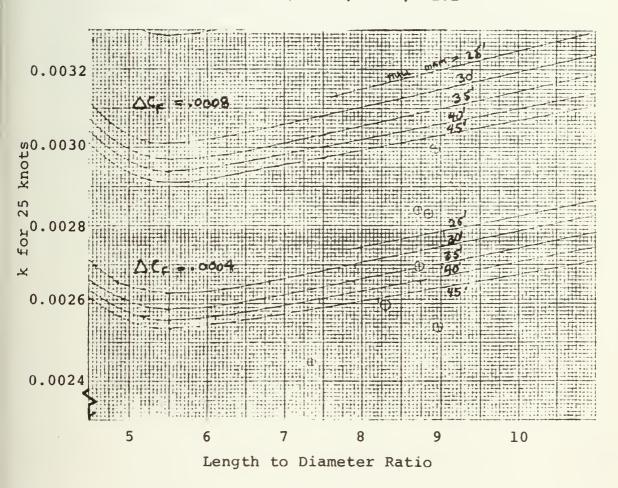
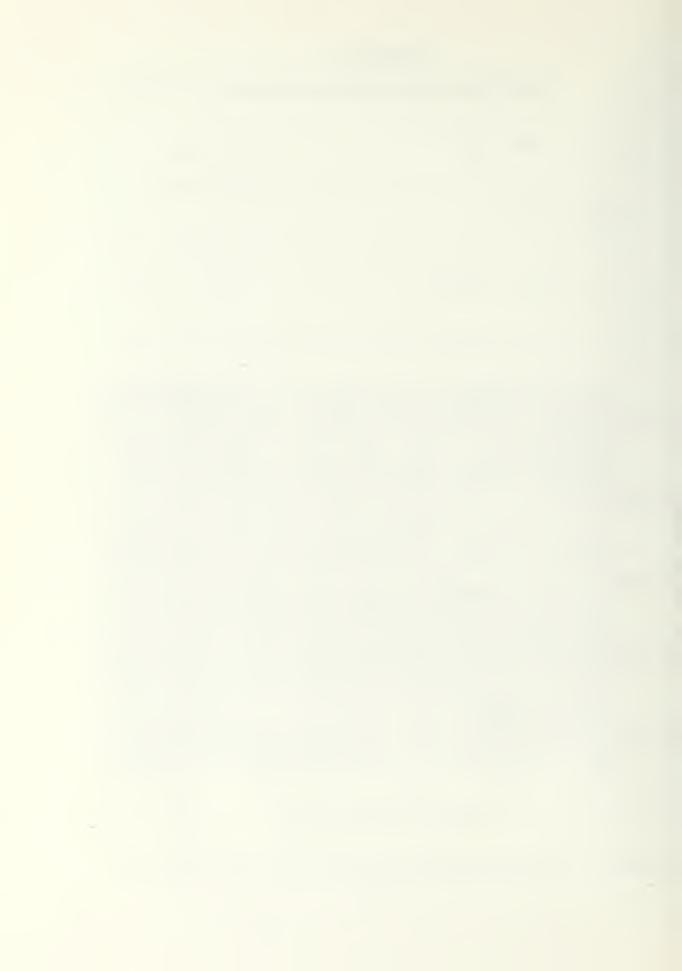


FIGURE 15 - Powering Coefficient k for 25 knots (ref. 10)



| VEHICLE MASS | DIAMETER | k |
|--------------|----------|----------|
| 1000 | 19.1 | 0.002825 |
| 2000 | 24.1 | 0.00279 |
| 5000 | 32.7 | 0.00272 |
| 10000 | 41.1 | 0.002665 |

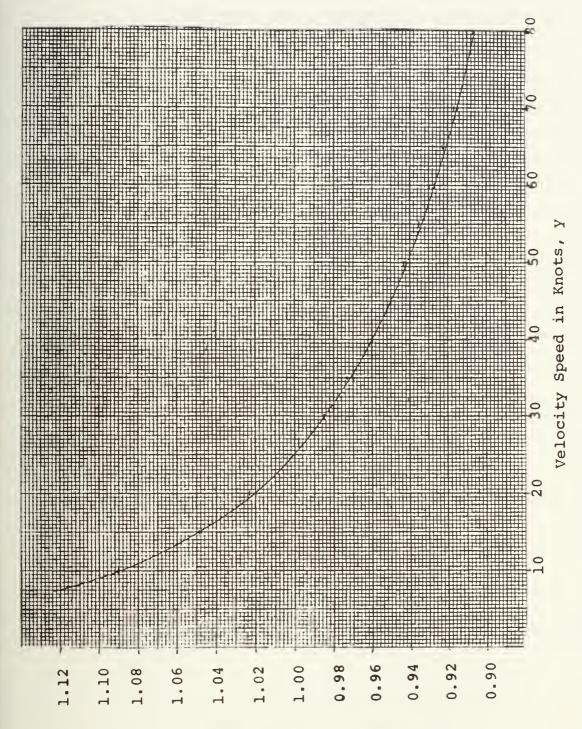
Since k is based on a 25 KT submarine, a factor P9 is used to correct for submarine velocity. (See Figure 17)

The equation used to estimate Figure 17 is:

$$P9 = 1 + 0.2 LOG_{10} (25/y)$$

$$k = P9 \cdot k_{25kts}.$$





SPEED CORRECTION FACTOR P9 vs. VELOCITY, y (Ref. 10) 16 FIGURE



APPENDIX II

FUEL CELL SHAFTING AND PROPELLER CALCULATIONS (Ref 10)

| INPUTS | VEHICLE MASS | DIAMETER (ft) | LENGTH SHAFT |
|--------|--------------|---------------|--------------|
| | 1000 | 12.73 | 34.4 |
| | 2000 | 16.04 | 43.3 |
| | 3000 | 21.77 | 58.8 |
| | 10000 | 27.43 | 74.1 |

Propeller Equations - fully submerged fixed pitch partially cavitating propeller

Weight = 6.5 x (diameter)³ (lbs)
RPM =
$$\{50 \text{ x } [(SHP)^{0.2}/(1.02 \text{ x diameter})]\}^{5/3}$$

Shafting Equations -

Intermediate Shafting
$$D_{O} = \frac{16T}{\pi S_{S}} \times \frac{1}{(1-\zeta^{4})} = \text{Shaft outer diameter - inches [4.14]}$$

T = Design Torque - lb x in

 $S_s = Shear stress due to torsion taken at 10,000 lb/in²$ lb/in²

$$\zeta = D_i/D_o = 0.1098 \text{ Log}_{10}^T + 0.2999$$
(Based on linear approximation to minimum shaft weight, per figure 15 of [4.14])



 $\zeta \leq 0.98$

D; = Shaft inner diameter - inches

$$L_{\text{max}}^2 = \frac{6\pi \text{ D}_{\text{O}}}{n} \sqrt{\frac{\text{gE}}{\rho_{\text{S}}}} (1+\zeta^2) = \text{Max shaft segment length} - \\ \text{inches [4.14] (To insure } \\ \text{operation below 1st critical } \\ \text{speed)}$$

n = rotational speed - rpm

 $g = acceleration of gravity - 386 in/sec^2$

 $E = Young's Modulus - 30 \times 10^6 lb/in^2$

 ρ_s = density of steel - .283 lb/in³

Propeller Shafting

same as above except:

ζ < 0.67

Intermediate and Propeller Shafting

W_{C&m} = coupling, bearing, lubrication and miscellaneous
 weights per shaft segment - lbs
= 6.6 x 10⁻⁴ x T + 5 x D_o x L/12 [4.14]
 L = shaft segment length - inches

$$W_{\text{shaft}} = .283 \times \frac{\pi}{4} \times (D_0^2 - D_1^2) \times L$$



Fuel Cell Group 2 Shafting and Propeller Weight

| VEHICLE MASS | EQUATION |
|--------------|--|
| 1000 | W(1) = 0.0001964 [SHP] + 9.43 |
| 2000 | W(2) = 0.0002532 [SHP] + 18.15 in (TONNES) |
| 5000 | W(3) = 0.0003451 [SHP] + 44.00 |
| 10000 | W(4) = 0.0005436 [SHP] + 82.30 |



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                                                                                                                                                                      H1=13888*((W1*0.9842) 0.57)
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APPENDIX IV. - LISTING OF FUEL CELL SUBMARINE PROGRAM

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Thesis C74745 171160

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Assessment of fuel cells and pressurized water nuclear reactors for submarine propulsion.

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